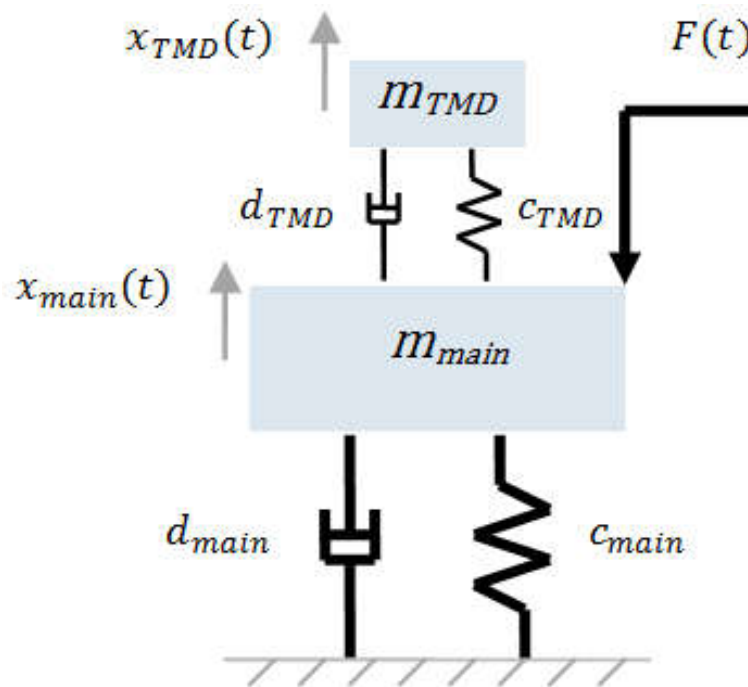


# Tuned Mass Damper Design for Attenuating Vibration

## Introduction

A mass-spring-damper is disturbed by a force that resonates at the natural frequency of the system. This application calculates the optimum spring and damping constant of a parasitic tuned-mass damper that the minimizes the vibration of the system.

The vibration of system with and without the tuned mass-spring-damper is viewed as a frequency response, time-domain simulation and power spectrum.



> restart : with( DynamicSystems ) : with( plots ) : with( SignalProcessing ) :

> params := [  $m_1 = 1.764 \cdot 10^5$ ,  $k_1 = 3.45 \cdot 10^7$ ,  $b_1 = 1.531 \cdot 10^5$ ,  $m_2 = 8165$  ] :

## Derive Expressions for the Optimum Spring and Damping Constant of the Tuned Mass Damper

Mass ratio

>  $\mu := \frac{m_2}{m_1}$  :

Natural frequency of tuned mass damper

$$> \omega_2 := \sqrt{\frac{k_{2\_calc}}{m_2}} :$$

Natural frequency of main system

$$> \omega_1 := \sqrt{\frac{k_1}{m_1}} :$$

Hence the natural frequency in rad s<sup>-1</sup> is

$$> \text{eval}(\omega_1, \text{params})$$

$$13.98492872$$

(2.1)

Ratio of natural frequencies

$$> \alpha := \frac{\omega_2}{\omega_1} :$$

Optimum ratio of natural frequencies

$$> \alpha_{opt} := \frac{1}{1 + \mu} :$$

Hence the optimum spring constant of the tuned mass-spring-damper

$$> k_{2\_calc} := \text{solve}(\alpha = \alpha_{opt}, k_{2\_calc})$$

$$k_{2\_calc} := \frac{m_1 k_1 m_2}{(m_1 + m_2)^2}$$

(2.2)

Damping Ratio

$$> z := \frac{b_{2\_calc}}{2 m_2 \omega_2} :$$

Optimum damping ratio

$$> z_{opt} := \sqrt{\frac{3\mu}{8(1+\mu)^3}} :$$

Hence the optimum damping constant of the tuned mass-spring-damper

$$> b_{2\_calc} := \text{solve}(z = z_{opt}, b_{2\_calc})$$

$$b_{2\_calc} := \frac{1}{2} \sqrt{6} \sqrt{\frac{m_2}{m_1 \left(1 + \frac{m_2}{m_1}\right)^3}} m_2 \sqrt{\frac{m_1 k_1}{(m_1 + m_2)^2}}$$

(2.3)

$$> k_{2\_calc} := \text{eval}(k_{2\_calc}, \text{params});$$

$$k_{2\_calc} := 1.458730861 \cdot 10^6$$

(2.4)

$$> b_{2\_calc} := \text{evalf}(\text{eval}(b_{2\_calc}, \text{params}));$$

$$b_{2\_calc} := 26869.77094$$

(2.5)

Full list of parameters

$$\text{> } \mathit{params}_{\mathit{tuned}} := \left[ m_1 = 1.764 \cdot 10^5, k_1 = 3.45 \cdot 10^7, b_1 = 1.531 \cdot 10^5, m_2 = 8165, k_2 = k_{2\_calc}, b_2 = b_{2\_calc} \right]:$$

$$\text{> } \mathit{params}_{\mathit{nottuned}} := \left[ m_1 = 1.764 \cdot 10^5, k_1 = 3.45 \cdot 10^7, b_1 = 1.531 \cdot 10^5, m_2 = 0, k_2 = k_{2\_calc}, b_2 = b_{2\_calc} \right]:$$

## ▼ Equations of Motion for the Entire System

Equation of motion for the whole system

$$\text{> } \mathit{de} := m_2 \left( \frac{d^2}{dt^2} x_2(t) \right) = -k_2 (x_2(t) - x_1(t)) - b_2 \left( \frac{d}{dt} x_2(t) - \frac{d}{dt} x_1(t) \right),$$

$$m_1 \left( \frac{d^2}{dt^2} x_1(t) \right) = -k_1 x_1(t) - b_1 \left( \frac{d}{dt} x_1(t) \right) - k_2 (x_1(t) - x_2(t)) - b_2 \left( \frac{d}{dt} x_1(t) - \frac{d}{dt} x_2(t) \right) + F(t):$$

$$\mathit{ic} := x_1(0) = 0, D(x_1)(0) = 0, x_2(0) = 0, D(x_2)(0) = 0:$$

$$\text{> } \mathit{sys} := \text{DiffEquation}([ \mathit{de}, [F(t)], [x_1(t)] ]):$$

## ▼ Frequency Response

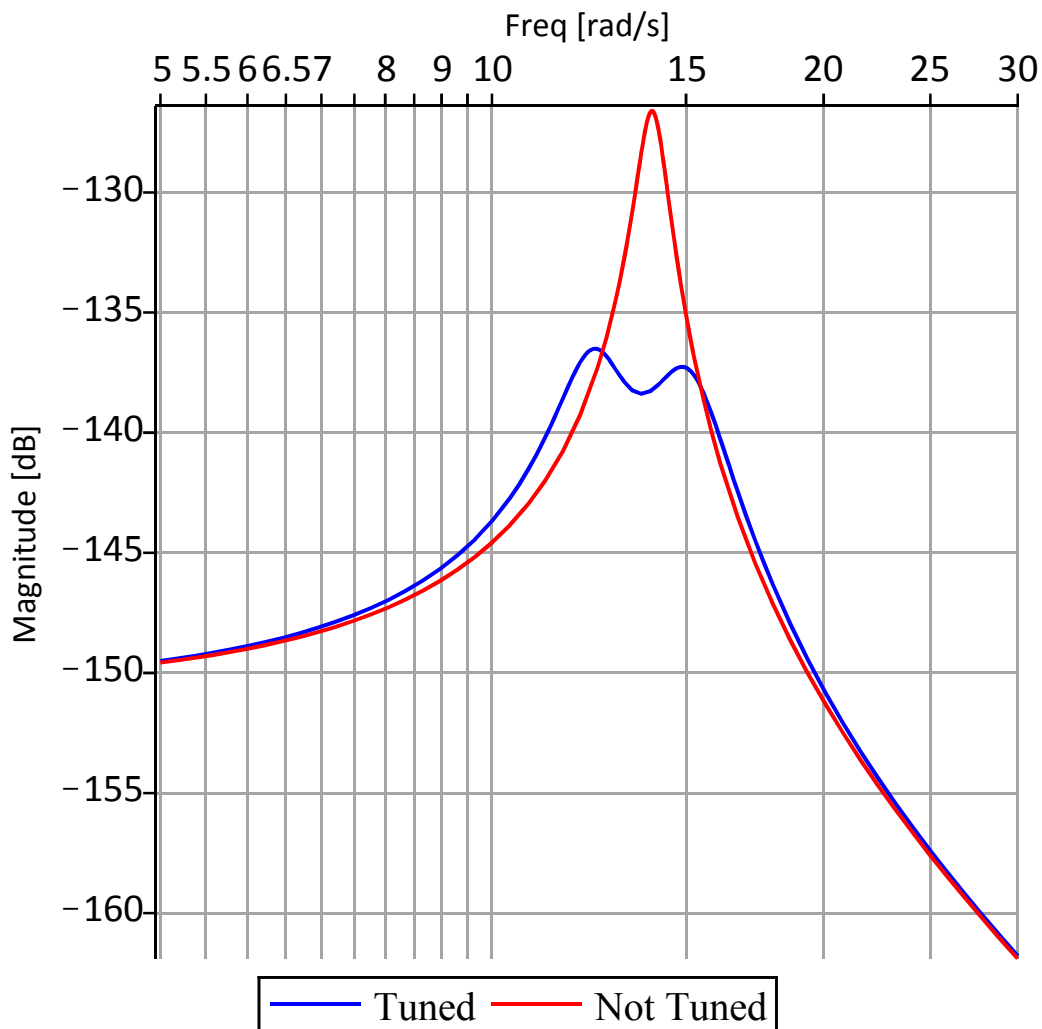
Response with Tuned Mass Damper

$$\text{> } \mathit{p1} := \text{MagnitudePlot}(\mathit{sys}, \text{range} = 5..30, \text{parameters} = \mathit{params}_{\mathit{tuned}}, \text{color} = \text{blue}, \text{legend} = \text{"Tuned"}):$$

Response with no Tuned Mass Damper

$$\text{> } \mathit{p2} := \text{MagnitudePlot}(\mathit{sys}, \text{range} = 5..30, \text{parameters} = \mathit{params}_{\mathit{nottuned}}, \text{color} = \text{red}, \text{axesfont} = [\text{Calibri}], \text{labelfont} = [\text{Calibri}], \text{legend} = \text{"Not Tuned"}):$$

$$\text{> } \text{display}(\mathit{p1}, \mathit{p2})$$



## ▼ Dynamic Response

Assuming that the main system is perturbed at its natural frequency

>  $f := \text{eval}(\omega_1, \text{params})$

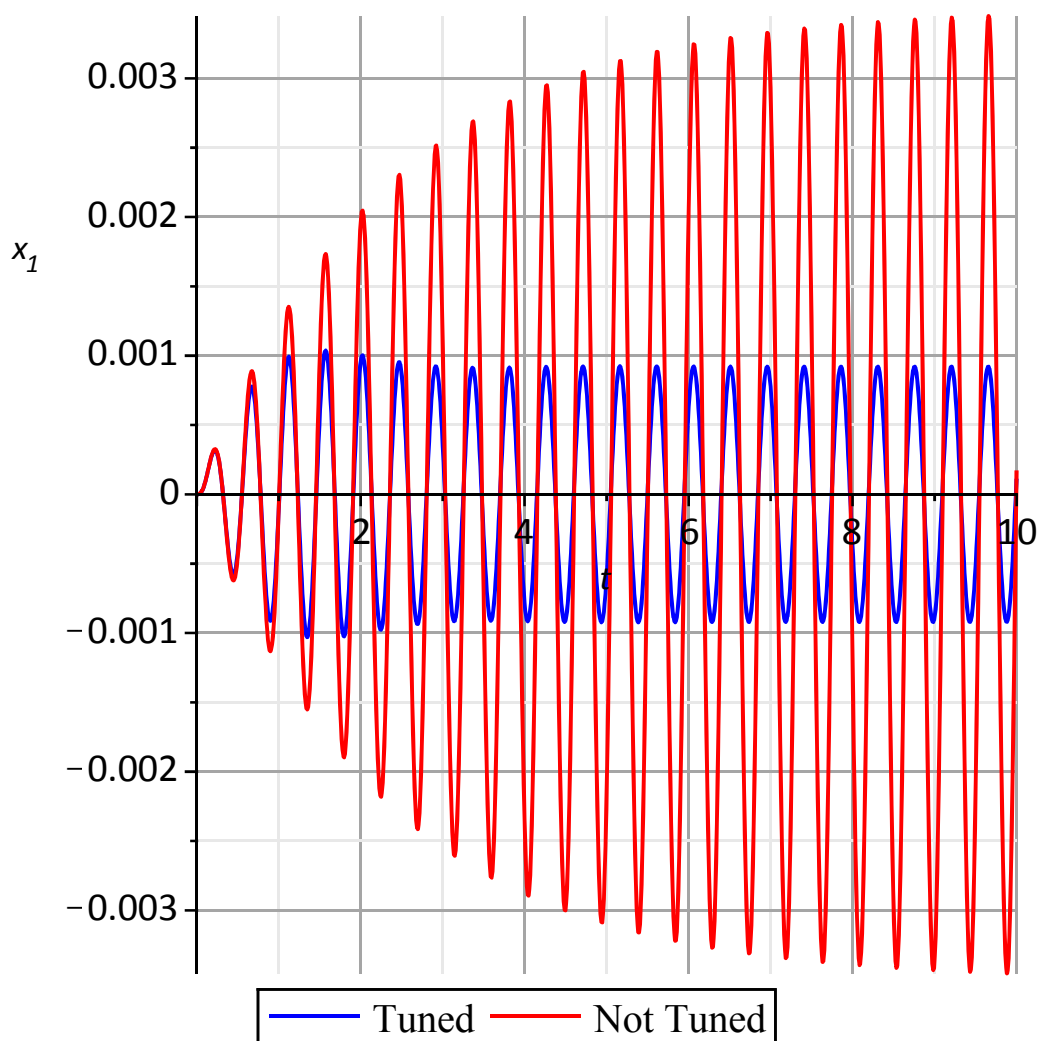
$f := 13.98492872$

(5.1

>  $p1 := \text{ResponsePlot}(\text{sys}, 7500 \sin(f \cdot t), \text{parameters} = \text{params}_{\text{tuned}}, \text{color} = \text{blue}, \text{numpoints} = 2^{10}, \text{legend} = \text{"Tuned"}) :$

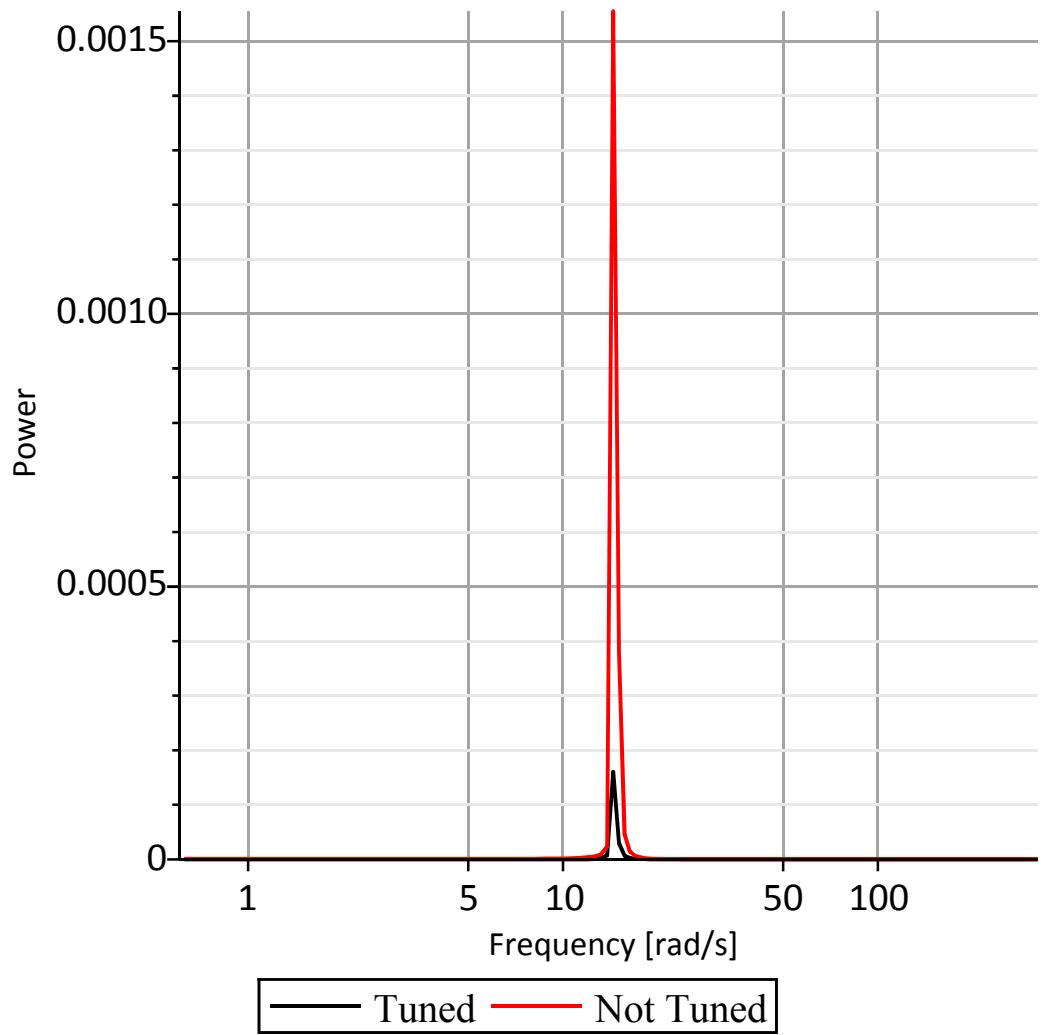
>  $p2 := \text{ResponsePlot}(\text{sys}, 7500 \sin(ft), \text{parameters} = \text{params}_{\text{nottuned}}, \text{color} = \text{red}, \text{numpoints} = 2^{10}, \text{legend} = \text{"Not Tuned"}) :$

>  $\text{display}(p1, p2, \text{axesfont} = [\text{Calibri}], \text{labelfont} = [\text{Calibri}], \text{axesfont} = [\text{Calibri}], \text{gridlines})$



## ▼ Power Spectrum

- > tunedResponseData := plottools:getdata(p1)[3]:
- > notTunedResponseData := plottools:getdata(p2)[3]:
- > samplingRate :=  $\frac{1}{\text{tunedResponseData}[2,1] - \text{tunedResponseData}[1,1]}$ :
- > psTuned := PowerSpectrum(FFT(tunedResponseData[.,2])):
- > psNotTuned := PowerSpectrum(FFT(notTunedResponseData[.,2])):
- > psPlot1 := pointplot( $\left[ \text{seq}\left(\left[\frac{i \cdot \text{samplingRate}}{2^{10}} \cdot 2 \cdot \text{Pi}, \text{psTuned}[i]\right], i = 1 .. \frac{2^{10}}{2}\right) \right]$ , connect = true, legend = "Tuned",  
gridlines):
- psPlot2 := pointplot( $\left[ \text{seq}\left(\left[\frac{i \cdot \text{samplingRate}}{2^{10}} \cdot 2 \cdot \text{Pi}, \text{psNotTuned}[i]\right], i = 1 .. \frac{2^{10}}{2}\right) \right]$ , connect = true, color = red, legend  
= "Not Tuned"):
- > display(psPlot1, psPlot2, axis[1] = [mode = log], axesfont = [Calibri], labels = ["Frequency [rad/s]", "Power"],  
labeldirections = [horizontal, vertical], labelfont = [Calibri])



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